

1 **Effects of PODE/diesel blends on particulate matter emission and particle**
2 **oxidation characteristics of a common-rail diesel engine**

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24 Abstract: The current study focused on the particulate matter emissions of polyoxymethylene dimethyl
25 ethers (PODE)/diesel blends with PODE blending ratios of 0, 10%, 20% and 30% have been experimentally
26 investigated in a common-rail engine. The influences of PODE blending ratio on the smoke emission,
27 particle size distribution and particle oxidation characteristic are discussed. Results show that the addition of
28 PODE in diesel fuel can effectively reduce smoke emission and its decreasing range becomes larger with
29 increasing PODE blending ratio. With the increment in PODE blending ratio, the particle concentration
30 distribution moves towards the direction of small particle size, and the total particle number concentrations
31 decrease. Besides, the peak values of particle number concentration, surface area concentration, and volume
32 concentration are all decreased. Adding PODE in diesel fuel increases the soluble organic fraction (SOF)
33 content of particles, rises maximum weight loss rate of particles, and lower the peak temperature of particles.
34 Also, the activation energy of pyrolysis reaction of particles decreases, which indicates that the oxidation of
35 particles becomes easier as PODE blending ratio increases. The apparent morphology of particles was
36 measured by scanning electron microscope, and the results show that the morphology of particles sample are
37 mostly chain like or flocculent.

38
39 Keywords: Diesel engine; Polyoxymethylene dimethyl ethers; Particulate matter emission; Particle oxidation;
40 Particle number

1. Introduction

The diesel engines, due to their higher power performance, efficiency and durability, have been widely employed in power generation and transportation for more than one hundred years. The increasing thermal efficiency and simultaneously reducing emissions for diesel engines are the primary objectives owing to fossil fuel shortage and environmental pollutions [1-3]. Particulate matter (PM) is the most prominent emission pollutant from diesel engines. In recent years, the total mass of PM emissions has been greatly reduced by various advanced combustion concepts and after-treatment technologies. Moreover, the focus of PM emission becomes the exhaust gas particle number concentration and size distribution. Diesel engine exhaust particle size distribution divided by nucleation-mode particles (diameter of the particle, $5 \text{ nm} < D_p < 50 \text{ nm}$), accumulation-mode particles ($50 \text{ nm} < D_p < 1000 \text{ nm}$) and coarse-mode particles ($D_p > 1000 \text{ nm}$) [4,5]. Nucleation-mode particles are constituted by that particulate core, formed by solid carbon particles combined with sulfuric acid vapor solid converted by part of the sulfur in diesel, adsorbs volatile organic compounds during the in-cylinder combustion process. Accumulation-mode particles are composed of nucleation-mode particles that are further aggregated into agglomerates and coagulated with semi-volatile substances such as sulfuric acid and HC [6].

Several studies confirmed that oxygenated fuel, playing an oxygen self-supplying role in the combustion process, can effectively reduce soot and particle emissions [7-10]. Polyoxymethylene dimethyl ethers (PODE) with no C-C bond, is a promising novel fuel for PM emissions decreased due to the low viscosities and pour points, low distillation temperatures, high oxygen contents and high cetane numbers [11,12]. The production costs of PODE are lower than that of hydrocarbons, especially in China, where the coal-based C_1 compounds like methanol and formaldehyde are oversupplied and low-cost commercial production of PODE has been realized [13,14]. There has been a great deal of studies related to combustion performance and emissions of diesel engine with PODE [15,16]. Pellegrini et al. [17] investigated the effects of blending

70 7.5% PODE into diesel on the emissions in a three cylinder, four-stroke direct injection (DI) Euro III
71 light-duty diesel engine. Their results indicated that the smoke and PM emissions were reduced by 32% and
72 13%, but the polycyclic aromatic hydrocarbon (PAH) emissions and particles ($D_p < 30\text{nm}$) marginally
73 increased. Liu et al. [18] studied the effect of the addition of 15% and 25% PODE (in volume) on
74 performance and exhaust emissions in a four-cylinder, turbocharged, heavy-duty, DI diesel engine. Their
75 results showed that the NO_x-soot trade-off relationship can be dramatically improved by fueling
76 diesel/PODE blends. The weighted results over the World Harmonized Stationary Cycle (WHSC) indicated
77 that the raw soot emission of blending 25% PODE into diesel can meet the Euro VI soot emission standard
78 when the weighted NO_x is controlled at 2.7 g/(kW·h).

79 Particle emissions represent a critical issue in particular for diesel engines blended with PODE, because
80 of their effect on human health. In order to make it clear of the source of PM emitted from a diesel engine
81 blended with PODE, lots of studies have explored the influences of PODE on particles [19,20]. Barro et al.
82 [21] investigated the performance of a research single cylinder heavy-duty 4-stroke diesel engine fueled with
83 PODE/diesel blends (20% and 80% PODE in volume) and found that the particle emissions using PODE
84 showed high number concentration (compared with pure diesel) in a size range mainly below 20 nm. Their
85 results reported that there was a bimodal distribution in the particle emission of PODE/diesel fuel. Moreover,
86 the peak value of small particle size increased and the peak value of large particle size decreased with the
87 proportion of PODE increasing. Recent evidence [22,23] indicates that PODE can be blended with diesel in
88 various ratios and direct application of PODE can reduce soot and PM emissions without changing the
89 existing diesel engine structure. Nevertheless, previous studies paid little attention on the effects of PODE
90 blending ratio on particle size distribution and particle oxidation characteristics in diesel engine, and further
91 investigation about them is necessary. Moreover, there are no enough research about the microstructures for
92 particles of blending PODE. From this view points of particles reduction, it is necessary to carry out the

93 research on the microstructures for particles.

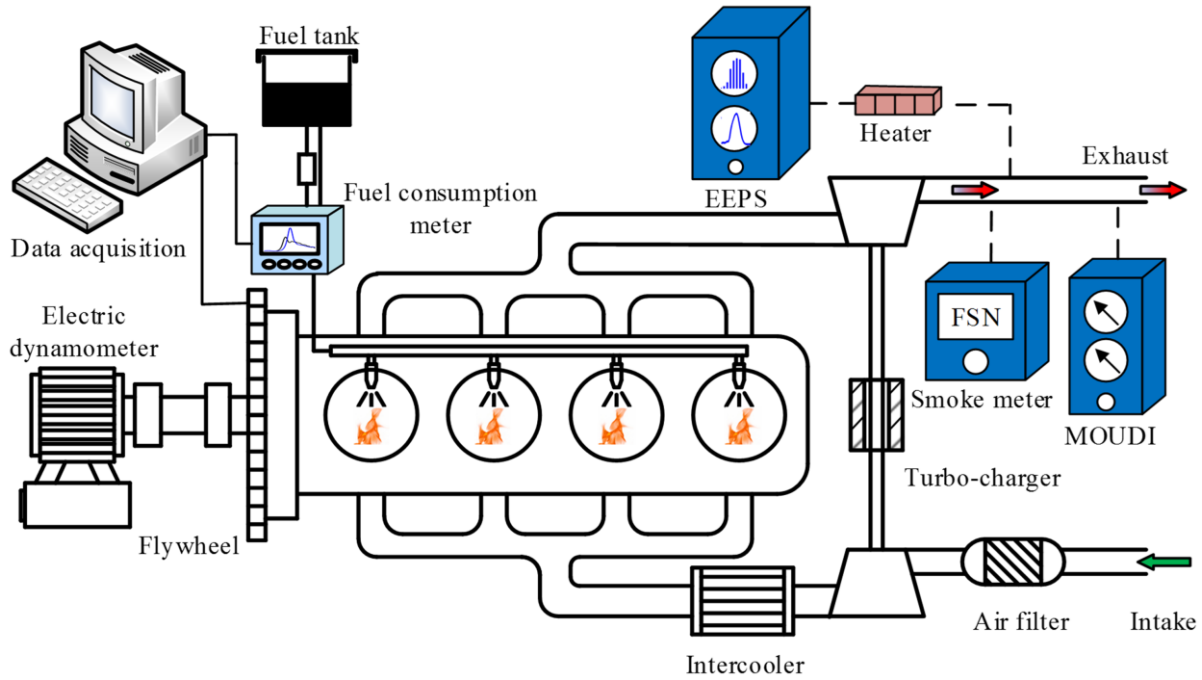
94 In current study, the exhaust PM emissions of pure diesel and PODE/diesel blends were systematically
95 explored on a 4-cylinder turbocharged intercooled common-rail diesel engine. Firstly, the effects of PODE
96 blending ratios on the exhaust particle number concentration, surface area concentration, and volume
97 concentration were discussed with a particle size spectrometer. Secondly, the oxidation characteristics of
98 particles were studied by thermogravimetric analysis (TGA), and particle activation energy was calculated
99 using the Coast-Redfern integral method. Finally, the microstructures of exhaust particles and smoke
100 emissions were analyzed using scanning electron microscopy and smoke meter, respectively.

101 **2. Experimental setup and procedure**

102 **2.1. Experimental apparatus**

103 The experimental engine employed for the current study was a four cylinder, four-stroke, turbocharged,
104 intercooled, common-rail diesel engine. Fig. 1 shows schematic diagram of the experimental setup. The
105 diesel engine structure and control strategy remain unchanged, and the main engine specifications are listed
106 in Table 1. In the experiment, the engine was controlled by an electrical dynamometer (AVL 504/4.6 SL)
107 coupled with an auto-testing and control system (AVL PUMA OPEN), which adjusts the torque output and
108 keeps the speed constant. The instantaneous blended fuel consumption rate was measured using an
109 intelligent fuel consumption meter (AVL 735s). The smoke emission was measured using a filter-type smoke
110 meter (AVL 415s), which detects soot particles in diesel engine exhaust through the blackness of filter paper
111 [24]. The particle size distribution and the number of that were measured using an engine exhaust particle
112 size spectrometer (TSI EEPS-3090), which can measure particle size from 5.6 nm to 560 nm [25].
113 EEPS-3090 used a dilution sampling scheme to measure particle concentration. The dilution ratio of the first
114 stage dilution system was set to 100:1, and the secondary dilution ratio was set to 2:1, thus the total dilution
115 ratio was 200:1. Moreover, the EEPS, the heating temperature of the dilution system controlled to 120°C,

116 can detect a complete particle size distribution map within 0.1s and output the number of particles and
 117 particle size distribution of 32 channels simultaneously. During the testing, 100s was taken as a time period
 118 to calculate the average distribution map of particle size. The particle samples were collected using a particle
 119 grading sampling device (MSP MOUDI-100). The engine exhaust, diluted and cooled, was sampled at 30
 120 L/min for 30 minutes under the action of a vacuum pump and collected PM for analysis via $\Phi 47$ aluminum
 121 foil filter paper. The oxidation properties of PM samples with different test fuels were measured using a
 122 synchronous thermal analyzer (METTLER TGA110) [26].



123
 124 **Fig.1.** Schematic diagram of the experimental setup.

125 **Table 1** Specifications of experimental engine.

Item	Specification
Engine type	In-line, 4 cylinder, 4 valves, turbocharged, water cooler
Bore \times Stroke	108 mm \times 118 mm
Displacement	4.32 L
Compression ratio	17.5 : 1
Rated power	98 kW (2400 r/min)
Maximum torque	400 N·m (1400~1600 r/min)
Combustion chamber shape	Direct injection ω type
Fuel injection system	Electronically controlled common-rail

126 **2.2. Test fuels**

The basic fuels adopted in the test were commercial ultra-low sulfur Euro VI diesel and PODE. Diesel fuel was produced by Sinopec Group, whose sulfur content is less than 5 ppm. Diesel is composed of numerous C₉-C₂₁ hydrocarbon species including paraffins, olefins, cycloparaffins and aromatics, and they are mainly n-paraffins, isoparaffins, naphthenes, aromatics and heteratomics [27]. PODE fuel was synthesized and separated by Qingdao Tongchuan Company, whose purity is 99%. PODE is a kind of acetal polymer with chain of the molecule of (-O-CH₂-) as the main chain, and its molecular formula is CH₃O(CH₂O)_nCH₃, where n stands for the polymerization degree and n≥1 [28]. PODE₃, PODE₄, PODE₅ and PODE₆ represent PODE fuels with polymerization degrees of 3, 4, 5 and 6, and their mass fractions are 39.3%, 25.1%, 21.3% and 14.3%, respectively in the test fuel of PODE. PODE was blended with diesel fuel at volume blending ratios of 0%, 10%, 20% and 30% to prepare PODE/diesel blends at room temperature and atmospheric pressure, and marked as D, P10, P20 and P30 fuels, respectively. The physical and chemical properties of the test fuels are given in Table 2. With the increment of PODE blending ratio, the cetane number, oxygen content and latent heat of vaporization for blend fuels increase, while the low heating value gradually decreases [29].

Table 2 Physicochemical properties of diesel, PODE and blended fuel.

Item	D	PODE	P10	P20	P30
Density at 15 °C (kg/m ³)	822	1050	845	868	890
Proportion of O (%)	0	42.7	5.9	11.4	16.7
Proportion of C (%)	86.5	44.1	81.2	76.2	71.4
Proportion of H (%)	13.49	8.78	12.9	12.4	11.82
Sulfur content (ppm)	5.0	0	4.4	3.8	3.2
Lower heating value (MJ/kg)	42.6	17.8	39.6	36.7	33.9
Cetane number	51.5	76	54	56.4	58.9
Latent heat of vaporization (kJ/kg)	250	359	263	277	289
End of boiling point (°C)	310	180	-	-	-
Kinematic viscosity at 20 °C (mm ² /s)	4.8	1.05	4.43	4.05	3.68

2.3. Test conditions and procedure

In the experiment, the effects of PODE/diesel blends (D, P10, P20 and P30) on PM emissions were analyzed under the engine loads of 0%, 25%, 50%, 75% and 100% at rated speed of 2400 r/min. Ambarish

145 et al. also selected the rated condition as the test condition in the exploration of combustion and emission
146 characteristics of diesel engine with alcohol addition [30]. Firstly, the smoke emission of four test fuels
147 versus engine loads was measured using the smoke meter, and the total particle number concentration was
148 analyzed using the engine exhaust particle size spectrometer correspondingly. Meantime, the effects of
149 PODE blending ratio on the particle diameter, particle surface area and particle volume distributions of 25%
150 and 100% loads were analyzed based on the measurement results. Finally, the PM samples of different test
151 fuels under rated condition for thermogravimetric testing and micro-morphology analysis were collected
152 using the particle grading sampling device [31]. The TGA of particle sample was conducted on a
153 thermogravimetric analyzer, which had built-in high precision electronic balance and temperature sensor.
154 The thermal weight loss process was recorded on-line by the data recording system. In each test, particle
155 sample with about 3mg was heated in the temperature range of 20~800°C, and the heating rates was set to
156 15°C/min. High purity N₂ and the air were taken as the protective gas and the reaction air, respectively, and
157 their flow rates were both set to a constant value of 50 mL/min. Thermogravimetric (TG) curves were
158 obtained by continually recording the mass loss with increasing temperature, and the derivative
159 thermogravimetric (DTG) curves were obtained by differentiating TG curves. The steady-state tests were
160 repeated at least twice to ensure that the results are repeatable within the experimental uncertainty. The
161 oxidation property of PM was obtained from the TG curve and the DTG curve, and their pyrolysis kinetic
162 parameters were calculated by the Coast-Redfern integral method. After surface gold spray treatment, the
163 microstructures of particle samples with four testing fuels were observed by a thermal field emission
164 scanning electron microscopy (SEM) (HITACHI S-4800). The resolutions of S-4800 were 1.0 nm (15 kV)
165 and 2.0 nm (1 kV), and its amplification range was 20~8×10⁵ times.

166 To improve measurement accuracy, experimentation with each test fuel was carried out three times and
167 their average values were considered for further calculation. During the testing process, there were several

chances for errors and uncertainties due to calibration, observation, instrument selection, working condition, environmental condition, etc. In order to prove the level of accuracy in measured and calculated readings, uncertainty analysis must be performed. Also, for normalizing the fixed/random errors occurred due to manufacturing syntax of various equipment, uncertainties were estimated in measured parameters using analytical method and the calculated uncertainties for various measured quantities were listed in Table 3.

Table 3 Accuracy and uncertainty of measured parameters.

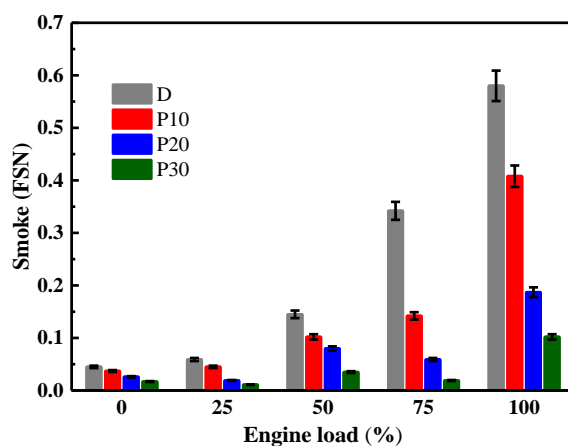
Instrument	Measured parameters	Range	Accuracy	Uncertainty (%)
Electric dynamometer	Speed	0~5000r/min	0.04%	-
	Torque	0~700N·m	0.4%	-
Fuel consumption meter	Fuel consumption	0~125kg/h	0.12%	0.5
Smoke meter	Smoke	0~10FSN	0.001FSN	0.5
Thermal analyze	Weight loss rate	0~10 mg	0.1μg	2.0
Engine Exhaust Particle Sizer	Particle size	5.6~560nm	0.1nm	-
MOUDI-100	Particle sampling	0.18~18μm	1nm	0.2

3. Results and discussion

3.1. Smoke emission

Fig. 2 reveals the effect of PODE blending ratio on smoke emission under different engine loads of 2400r/min. It can be observed from Fig. 2 that the addition of PODE into diesel fuel can effectively decrease the smoke emission, and the reduction obviously rises with the increase of PODE blending ratio and engine load. Compared with pure diesel, the smoke emission levels of P10, P20 and P30 are decreased by 29.7%, 67.8% and 82.4% at rated condition. Generally, in-cylinder soot formation of diesel engine is due to the local hypoxia and high-temperature fuel cracking caused by the uneven mixture of fuel and gas, which is mainly generated in the diffusion combustion stage. Because of the low viscosity and boiling point of PODE fuel, adding PODE in diesel fuel is conducive to the atomization and evaporation of the blended fuel, and also can improve the uniformity of fuel-gas mixture [32,33]. The addition of PODE will increase oxygen content of the blends, which plays a self-oxygen supply role during combustion, thus improving diffusion combustion, and promoting the post oxidation process of carbon smoke in the subsequent expansion and

187 exhaust stages. The results of chemical reaction kinetics calculation show that carbon atoms existing as C-O
188 bonds in PODE are difficult to participate in the free radical reactions of small-molecule formation. Thus,
189 the substitution of PODE for diesel reduces the generation amount of soot precursor PAH [29,34]. Moreover,
190 a large number of OH oxidation groups are formed in the early stage of PODE combustion, which can
191 directly react with ethylene or acetylene, thereby inhibiting the formation of PAH and the growth of soot
192 rings [35]. The above reasons lead to the reduction of exhaust smoke with the addition of PODE in diesel
193 fuel.



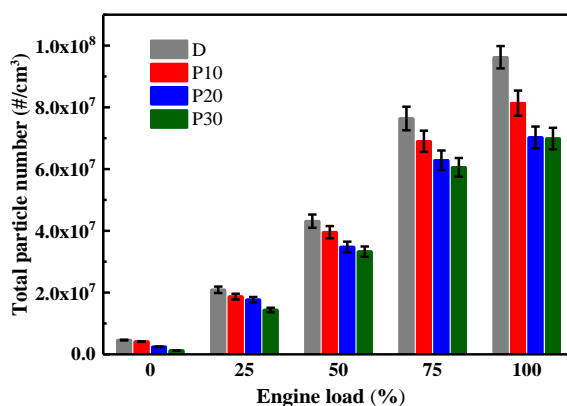
194
195 **Fig. 2.** Effects of PODE blending ratio on smoke emission for various loads.

196 **3.2. Exhaust particle number and particle size distribution**

197 In general, PAH adsorbed on the surface of carbon particles will damage human health. In terms of
198 harmfulness to human body and atmospheric environment, the quality of particle emission does not play a
199 decisive role, but the size distribution and number concentration of particles are more important. China VI
200 emission regulations for diesel vehicles clearly defines the emission limits of total particle number, which
201 have been carried out since 2019.

202 Fig. 3 presents the effect of PODE blending ratio on total particle number concentration under different
203 engine loads of 2400r/min. It can be observed that total particle number concentration of the blends rises
204 with increasing engine load whereas decreases with increasing PODE blending ratio, and diesel fuel is the
205 largest producer of total particle number in all conditions. Under rated condition, the total number of exhaust

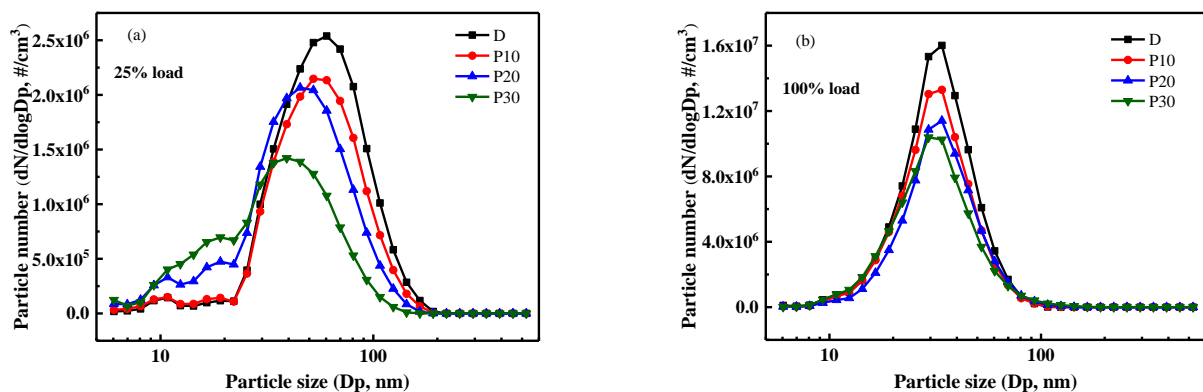
206 particles with P10, P20 and P30 fuels are decreased by 15.5%, 27.0% and 28.1% as compared with pure
 207 diesel, respectively. This is mainly due to the higher cetane number, lower sulfur and aromatics contents, and
 208 lower viscosity of PODE comparing to diesel fuel. With the addition of PODE in diesel fuel, the blends have
 209 higher cetane number and oxygen content, and the ignition timing is advanced and the burning speed
 210 improves rapidly [36], which helps to reduce particle emission. Because of the high carbon content and
 211 stable chemical structure of aromatics, it is easy to form soot particles during combustion. The blended fuel
 212 has less aromatics, so adding PODE to diesel fuel can reduce the exhaust particles of accumulation-mode for
 213 diesel engines [37]. The sulfur in the fuel will be discharged in the form of sulfate particles after combustion
 214 stage, thus the lower sulfur content of the blended fuel will help to reduce the sulfated particles of
 215 nucleation-mode [38,39]. The lower viscosity and boiling point of the blends can improve the interior fuel
 216 atomization inside the cylinder, which makes combustion of air-fuel mixture more sufficient, resulting in the
 217 reduction of the formation of nucleation-mode particles. All of these factors together make the total particle
 218 number emission decrease with the addition of PODE in diesel fuel.



219
 220 **Fig. 3.** Effects of PODE blending ratio on total particle number concentration for various loads.

221 Fig. 4 illustrates the effect of PODE blending ratio on particle number concentration distribution of the
 222 blends at 25% and 100% loads of 2400 r/min. It can be seen from Fig. 4(a) that at 25% engine load, the
 223 particle number concentration with particle size distribution appears a bimodal curve: the first peak occurs at
 224 10~20 nm, belonging to the nucleation-mode scope; the second obvious peak occurs at 50~70 nm, belonging

225 to the accumulation-mode scope. With the increment in PODE blending ratio, the peak concentration of
 226 accumulation-mode particles gradually decreases and the peak concentration of nucleation-mode particles
 227 gradually increases, while the whole distribution curve of particle number concentration moves to the
 228 direction of small particle size. As seen in Fig. 4(b), the particle number concentration distribution acts as a
 229 single peak presenting in the range of 30~40 nm under rated condition. The particle emission of the blends is
 230 mainly in the nucleation-mode region, and the number concentration of accumulation-mode particles is
 231 relatively low, besides the peak concentration of exhaust particles drops down with an increase in PODE
 232 blending ratio. Nucleation-mode particles are usually the primary carbon particles formed by nonuniform
 233 combustion in cylinder and the secondary particles formed by dilution and cooling nucleation of precursors
 234 such as HC and sulfuric acid, etc. And nucleating effect is affected by the specific surface area of particles,
 235 ambient humidity and temperature in the exhaust [40]. The concentration of accumulation-mode particles
 236 with large particle size decreases significantly with the addition of PODE into diesel fuel, which reduces the
 237 adsorption capacity of gaseous precursors (HC or sulfuric acid) on the surface of accumulation-mode
 238 particles and increases the nucleating effect during the exhaust and dilution processes, resulting in an
 239 increase of the number concentration of nucleation-mode particles.



240

241

Fig. 4. Effects of PODE blending ratio on particle number concentration distribution: (a) 25% load; (b) 100% load.

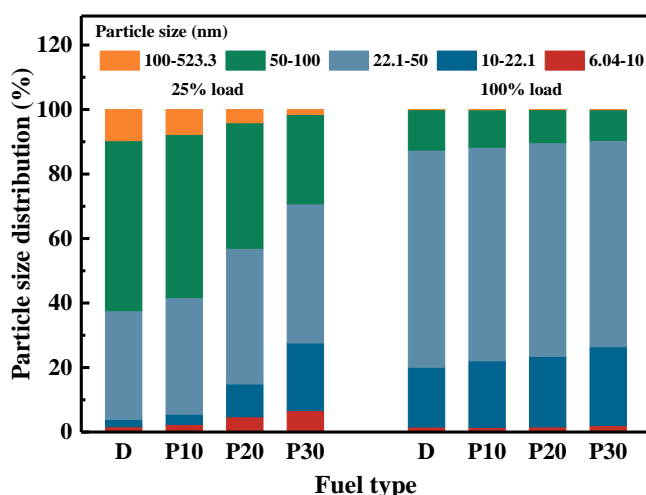
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Fig. 5 shows the proportion of particle size distribution from diesel exhaust versus PODE blending ratios,

243

which is obtained by summing the particle number concentration in different particle size segments in Fig. 4.

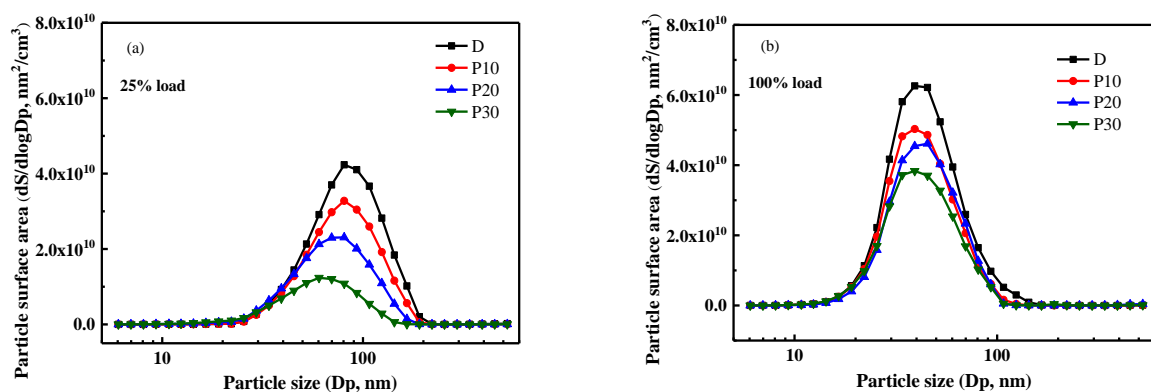
244 With the increase of PODE blending ratio, the proportion of ultra-fine particles ($D_p < 22$ nm) increases
 245 significantly, and the proportion of nucleation-mode particles increases correspondingly, while the
 246 proportion of accumulation-mode particles decreases gradually, especially at low load. At 25% load, the
 247 proportions of nucleation-mode particles for pure diesel, P10, P20 and P30 fuels are 37.6%, 41.7%, 57.0%
 248 and 70.8%, and the corresponding proportions of accumulation-mode particles are 62.4%, 58.3%, 43.0% and
 249 29.2% respectively. The reason for the increase in the number of nucleation-mode particles and the decrease
 250 in the number of accumulation-mode particles is that the blends have larger penetration distance, good
 251 evaporation and atomization characteristics [41,42]. The fuel concentration in the injection periphery and
 252 near-wall region is too lean to burn completely, which increases the generation of SOF, thus leading to the
 253 increase of nucleation-mode particles. The smoke emission of the blends is lower, and the adsorption
 254 capacity of the accumulation-mode particles for SOF is reduced, further leading to the increase of
 255 nucleation-mode particles [43]. Due to the self-contained oxygen of the blends, some accumulation-mode
 256 particles will be oxidized to nucleation-mode particles in post-combustion period [44], which reduces the
 257 proportion of accumulation-mode particles.



258 **Fig. 5.** Effects of PODE blending ratio on particle size contribution at 25% and 100% loads.

260 The larger the surface area of the particles is, the stronger the absorption capacity of organic components
 261 with high boiling point in exhaust gas is, the greater the toxicity is, the more likely it is to cause harm to

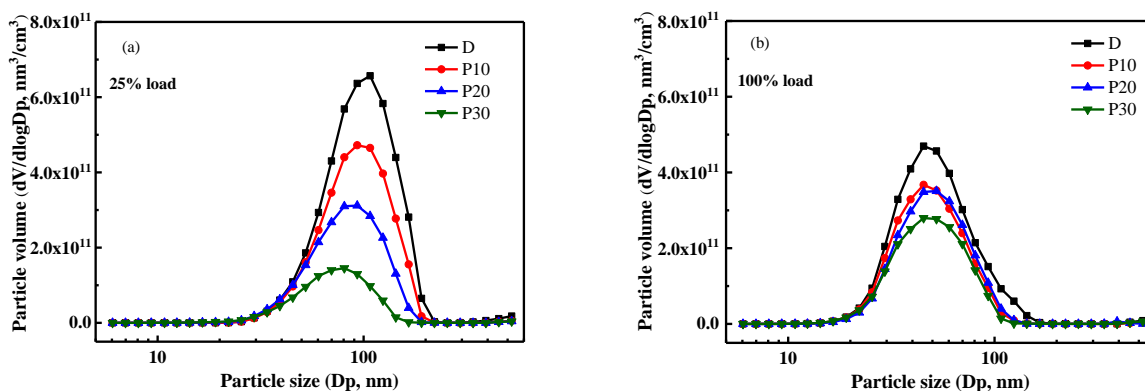
262 human body. Fig. 6 presents the effect of PODE blending ratio on particle surface area concentration
 263 distribution of the blends at 25% and 100% loads of 2400 r/min. As shown in Fig. 6(a), the varying curve of
 264 particle surface area concentration acts as a single peak morphology, mainly distributed in the
 265 accumulation-mode state, and the peak appears at about 80 nm. With the increase of PODE blending ratio,
 266 the peak value of particle surface area concentration decreases obviously, and the peak position gradually
 267 shifts to the left. This shows that adding PODE to diesel fuel can significantly reduce the emission of
 268 accumulation-mode particles, which is consistent with the results in Fig. 4(a), and the shift of peak position
 269 to the left indicates that the proportion of ultrafine particles increases, which is consistent with the results in
 270 Fig. 5. The change trend presented in Fig. 6(b) is similar to that of Fig. 6(a), but it is mainly distributed in
 271 the nucleation-mode state. The peak appears at about 40 nm, and the overall peak value of 100% load is
 272 higher than that of 25% load. It is mainly due to the high combustion temperature and exhaust temperature
 273 at high load, which makes the accumulation-mode particles oxidize to form nucleation-mode particles, so
 274 that both the peak value of particle number concentration and the proportion of ultra-fine particles become
 275 larger [4, 45], which are identical with the results presented in Fig. 4.



276
 277 **Fig. 6.** Effects of PODE blending ratio on particle surface area concentration distribution: (a) 25% load; (b) 100% load.

278 Fig. 7 reveals the effect of PODE blending ratio on particle volume concentration distribution of the
 279 blends at 25% and 100% loads of 2400 r/min. It can be seen that the particle volume concentration decreases
 280 with the increase of PODE blending ratio, and its distribution presents a single peak. The peak position of

281 particle volume concentration moves to large particle size compared with the peaks of particle number and
 282 particle surface area concentrations. Particle volume concentration peaks of 25% and 100% loads appear at
 283 100nm and 50nm respectively, and particle volume concentration of 100% load is lower than that of 25%
 284 load. This is because a small number of accumulation-mode particles will inhibit the adsorption and
 285 coagulation of nucleation-mode particles, resulting in an increase of nucleation-mode particles with engine
 286 load, and the influence of small particles on particle volume concentration distribution is so little [46,47].
 287 The results also show that the addition of PODE in diesel fuel can reduce particle volume concentration,
 288 which makes it easier for diesel exhaust particles to sink and shortens the residence time in the air.



289

290 **Fig. 7.** Effects of PODE blending ratio on particle volume concentration distribution: (a) 25% load; (b) 100% load.

291 3.3. Particle oxidation property

292 Fig. 8 illustrates the particle TG and DTG curves of the blends with different PODE blending ratios at
 293 rated condition and in air atmosphere. It can be observed from Fig. 8(a) that the heating process of particles
 294 is mainly divided into three reaction stages: the first stage is the water evaporation stage (40~110 °C); the
 295 second stage is the SOF volatilization and oxidation (120~250 °C); the third stage is the particles oxidation
 296 (400~650 °C). As shown in Fig. 8(b), the weight loss rate increases significantly when the TGA temperature
 297 is higher than 120 °C, which is the oxidation and volatilization of high volatile SOF. However, as the TGA
 298 temperature reaches 200 °C, the weight loss rate gradually decreases, which is the oxidation and
 299 volatilization of low volatile SOF. There are two obvious peaks of weight loss rate in the DTG curve. The

300 first peak of weight loss rate appears at about 160 °C, which indicates the pyrolysis process of SOF
301 components in the air atmosphere. The heat volatilization of SOF is accompanied by the oxidation reaction
302 under the action of O₂, which makes the weight loss rate of particles increase significantly. The second peak
303 of weight loss rate appears near 580 °C, which indicates the mass change process of soot oxidation. The data
304 show that soot components in the particles of diesel, P10, P20 and P30 fuels account for 88.1%, 85.8%,
305 84.2% and 83.6%, and SOF components account for 8.7%, 11.6%, 12.9% and 13.6%, respectively. The peak
306 values of absolute weight loss rate are 0.41, 0.46, 0.48 and 0.52 %/°C, and the corresponding temperatures
307 of the peaks are 584, 580, 576 and 569 °C, respectively. It can be seen that with the increase of PODE
308 blending ratio, the mass fraction of SOF component increases, the mass fraction of soot component
309 decreases, the weight loss rate increases, and the peak temperature decreases. This is mainly due to the
310 increment in the oxygen content of the blends with addition of PODE, more oxygen-containing groups will
311 be adsorbed on the surface of particles after combustion, so the oxidation combustion rate of SOF
312 component becomes faster. Fuel self-contained oxygen also improved the oxygen deficient condition of
313 combustion in the cylinder. The amount of OH radicals generated in the initial combustion stage increase
314 [48], and the oxidation of OH reduces PAH and its precursor, inhibiting the growth of soot particles, thus the
315 emission of soot particles is less and the particle size is small. After adding PODE, the C-H bond far away
316 from oxygen-containing group in soot particles breaks first to form more organic carbon, which has stronger
317 oxidation characteristics than element carbon. As a result, once the temperature rises to a certain value, the
318 organic carbon will be oxidized and burned rapidly, resulting in an increase in the decline rate of soot mass
319 [49,50].

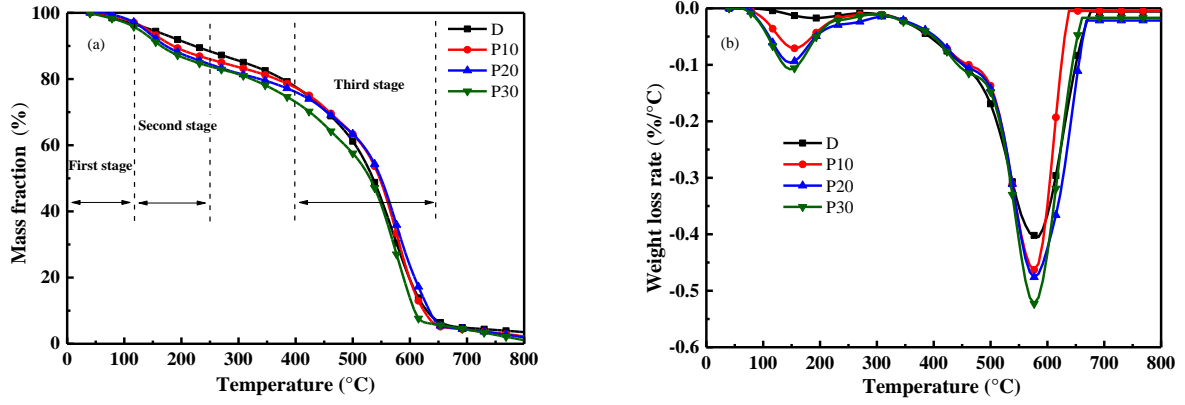


Fig. 8. TG and DTG curves of particle samples for various PODE blending ratios: (a) TG curve; (b) DTG curve.

Pyrolysis kinetics analysis of particle samples with the blends was carried out in the air atmosphere with Coats-Redfern method to obtain the reaction activation energy parameters of particles, which can be used to judge the difficulty of particle oxidation with different PODE blending ratios. Generally, activation energy refers to the minimum energy required for molecules to change from normal state to active state in chemical reactions, and the higher the activation energy is, the more energy is needed for pyrolysis. The equation for calculating the pyrolysis reaction of diesel exhaust particles by Coats-Redfern method is as follows:

$$G(\alpha) = \int_0^\alpha \frac{1}{(1-\alpha)^n} d\alpha = \int_0^T \frac{A}{\beta} e^{-\frac{E}{RT}} dT \quad (1)$$

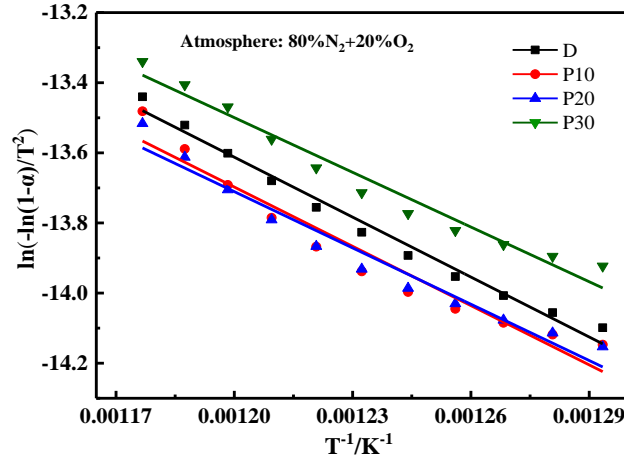
where α denotes the conversion rate of particle sample (%); A denotes the pre-exponential factor (min^{-1}); T denotes the reaction temperature (K); E denotes activation energy (kJ/mol); R denotes the ideal gas constant ($8.314\text{J}/(\text{K}\cdot\text{mol})$); β denotes the heating rate (K/min), and is set to 15 K/min in the experiment; n denotes the reaction order, and is often taken as 1 in the oxidation environment for diesel particles.

Sort out and simplify Eq. (1) to get Eq. (2):

$$\ln \left[\frac{-\ln(1-\alpha)}{T^2} \right] = \ln \left(\frac{AR}{\beta E} \right) - \frac{E}{RT} \quad (2)$$

Eq. (2) be regarded as one-variable linear equation with a slope of $-\frac{E}{R}$, an independent variable of $\frac{1}{T}$ and a dependent variable of $\ln \left[\frac{-\ln(1-\alpha)}{T^2} \right]$, and then E and A are can be calculated from the fitting straight line. Fig. 9 presents the pyrolysis characteristic curves of particle samples with four test fuels obtained using

338 fitting method, while the corresponding fitting equations and kinetic parameters of soot particles in the most
 339 intense reaction stage are listed in Table 4. Calculation results show that the fitting linear coefficients of all
 340 curves are greater than 0.95, which indicates that the ideal degree of fitting. The activation energy of particle
 341 pyrolysis reaction decreases with the addition of PODE in diesel fuel, and it decreases from 47.4 kJ/mol to
 342 43.2 kJ/mol as PODE blending ratio increases from 0 to 30%. The result is consistent with the trends that the
 343 weight loss rate of soot particles rises and the peak temperature reduces with blending PODE in Fig. 8(b).
 344 This is due to the increase of organic carbon content in soot particles under the action of oxygen-containing
 345 group after adding PODE to diesel fuel. In addition, the disorder degree of soot particles increases and the
 346 graphitization degree decreases with an increase in the proportion of ultrafine particles for the blends, which
 347 makes the particles easier to be oxidized. Therefore, the energy required in the particle pyrolysis process
 348 decreases, and the reaction activation energy decreases [51].



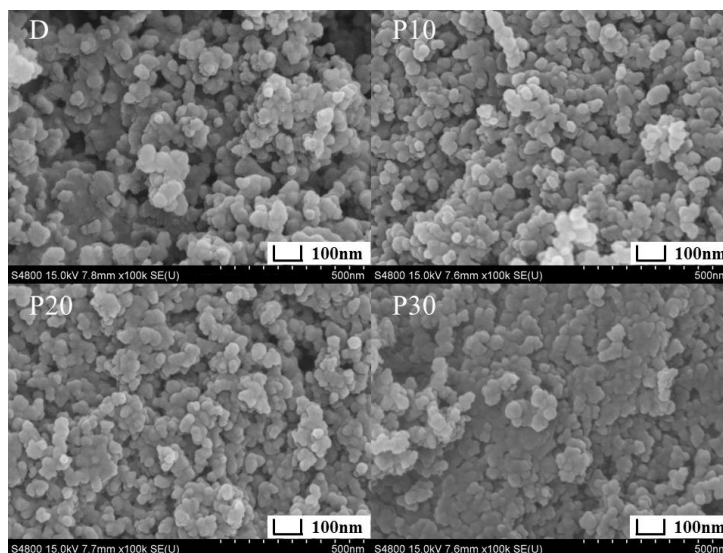
349
 350 **Fig. 9.** Fitting curves of particle pyrolysis characteristic with Coats-Redfern method.

351 **Table 4** Calculated kinetic parameters of soot particles.

Particle sample	Fitting equations	Regression coefficient / R^2	Activation energy E / (kJ/mol)	Pre-exponential factor A / (min^{-1})
D	$y=-5\ 698.65x-6.77$	0.9833	47.37	130.28
P10	$y=-5\ 636.16x-6.93$	0.9501	46.85	109.78
P20	$y=-5\ 352.90x-7.28$	0.9574	44.50	73.25
P30	$y=-5\ 201.29x-7.35$	0.9635	43.24	66.31

352 3.4. Particle microscopic morphology

353 Fig. 10 gives the microscopic morphology of particle samples with diesel, P10, P20 and P30 fuels under
354 rated condition characterized by means of SEM and magnified 100,000 times. It can be seen from the Fig.
355 10 that the particles morphology of the blends is similar to that of pure diesel, and the particle size of
356 individual particle is in the range of 30~50nm. Due to the high surface activity of fine particles, the
357 agglomeration is easy to occur between particles, and most of them are chain like or flocculent structure.
358 However, the difference between the blends and diesel fuel is that the particles size changes inconsistently,
359 also the single particle size decreases and the agglomeration degree between particles increases with an
360 increase in PODE blending ratio.



361
362 **Fig. 10.** SEM images of particle samples with the blends.

363 The SEM image of diesel exhaust particles has a typical fractal structure, and the growth mechanism of
364 particles determines its fractal structure characteristics. The agglomeration form of particles is mainly
365 divided into sedimentation under gravity, and the particle diffusion and Brownian motion in the turbulent
366 process. The fractal dimension can be used to quantitatively study the density of the particle geometric
367 structure. The larger the fractal dimension is, the higher the degree of overlap between the base carbon
368 particles is, and the more compact the particle structure is [52]. The self-similar fractal dimension, the
369 box-counting dimension, is applied to investigate the fractal characteristics of PODE/diesel blended fuel

370 particles in this study. The equation for calculating box-counting dimension is as follows:

$$371 \quad D_B = -\lim_{r \rightarrow 0} \frac{\lg N(r)}{\lg r} \quad (3)$$

372 where D_B denotes the box-counting dimension of the particle SEM image; r denotes the side length of the
373 square box; $N(r)$ denotes the number of square boxes that completely cover the particle pattern area.

374 The box-counting dimension of PODE/diesel particles was calculated by MATLAB software as follows:
375 firstly, the image processing is to read the original SEM image of particles; secondly, gray matrix of the
376 image is calculated; thirdly, the threshold of image binarization is calculated; fourth, the fitting equation (4)
377 is obtained by fitting the binary graph covered by a square box with side length r . The slope of the fitting
378 equation D_B is the fractal dimension.

$$379 \quad y = -D_B x + b \quad (4)$$

380 The binarization threshold of the SEM image and the relationship of $\lg N(r)$ - $\lg r$ of diesel particles are
381 obtained via calculating the box-counting dimension of the SEM image of pure diesel combustion particles.
382 The result of fitting the curve is shown in Fig. 11. It can be observed that the regression coefficient of the
383 fitted curve R^2 is 0.9959, and the box-counting dimension of diesel particles is 1.913. The box-counting
384 dimensions of P10, P20 and P30 fuel combustion particles can be calculated with the same method. The
385 fitting equations, regression coefficients and box-counting dimensions of the particles with different PODE
386 blending ratios are listed in Table 5. The results show that the $\lg N(r)$ - $\lg r$ relationship of P10, P20 and P30
387 particles are well fitted, and their box-counting dimensions are 1.936, 1.944 and 1.978, respectively. It can
388 be seen that the box-counting dimension of the particles gradually increases with PODE blending ratio,
389 which indicates that the degree of cluster between particles is gradually enhanced.

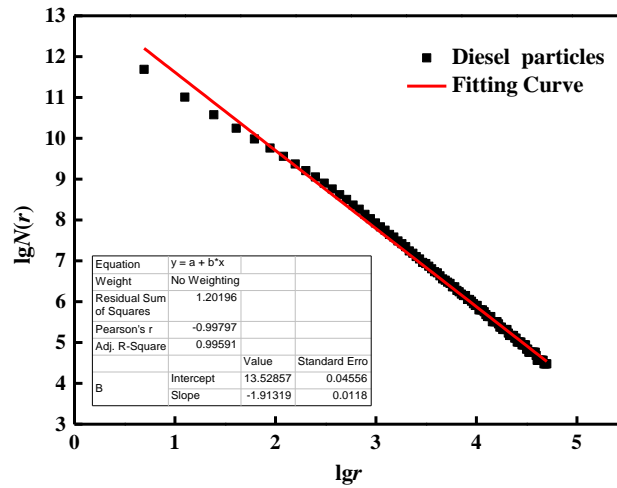


Fig. 11. The fitting curve of combustion particles with pure diesel.

Table 5 Box-counting dimension of PODE/diesel blends particles.

Particle sample	Fitting equations	Regression coefficient / R^2	D_B
D	$y = -1.913x + 13.529$	0.9959	1.913
P10	$y = -1.936x + 13.624$	0.9970	1.936
P20	$y = -1.944x + 13.650$	0.9970	1.944
P30	$y = -1.978x + 13.796$	0.9983	1.978

4. Conclusions

In current study, the particle size distribution and oxidation characteristics of exhaust PM were carried out on a common-rail engine with PODE/diesel blends at various PODE blending ratios by means of EEPS and TGA. The main conclusions are summarized as follows:

(1) PODE/diesel blends can effectively reduce the smoke emission for diesel engines and its decreasing range becomes larger with PODE blending ratio increasing. At rated condition, the smoke emissions of P10, P20 and P30 decrease by 29.7%, 67.8% and 82.4% respectively as compared to pure diesel.

(2) The addition of PODE in diesel fuel can significantly decrease the total particle number, especially at high load. Compared with pure diesel, the total particle number concentrations of P10, P20 and P30 at rated condition are reduced by 15.5%, 27.0% and 28.1%, respectively.

(3) As PODE blending ratio increases, the particle concentration distribution moves towards the direction of small particle size, the proportion of nucleation-mode particles increases, and the proportion of accumulation-mode particles decreases. Also, the peak values of particle number concentration, surface area

406 concentration and volume concentration decrease, and the particle size at the peak position decrease.

407 (4) With the increment in PODE blending ratio, the SOF content in exhaust particles increases, the
408 weight loss rates of soot particles and SOF component are increased, and the corresponding peak
409 temperatures are decreased. Meantime, the activation energy of particle pyrolysis is reduced, which indicates
410 that particle oxidation reaction becomes easier with adding PODE to diesel fuel.

411 (5) Particle microscopic morphology of the blends is similar to that of pure diesel, and the single particle
412 size is in the range of 30~50nm. The morphology of particles sample is mostly chain like or flocculent, and
413 the agglomeration degree between particles increases with the addition of PODE in diesel fuel.

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